# DEVELOPMENT OF TWO-PHASE FLOW SIMULATION USING SPH METHOD 

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#### Abstract

Gas entrainment is one of the major defects caused in the casting filling process. Since the particle method is a Lagrangian method that does not use a lattice, it can easily analyze large deformations and boundary movements, so it has the potential to be applied to gas defect prediction as a methodology. However, in the two-phase flow simulation including gas entrainment, the analysis fails when the gas / liquid density ratio becomes smaller than about $1 / 10$ in the conventional SPH. Therefore, two dimensional two-phase flow SPH methodology was developed. Then, authors extended the methodology to three dimensions that can be applied to gas entrainent during a die cast filling process.


## 1 INTRODUCTION

Gas entrainment is one of the major defects caused in the foundry filling process. Since the smoothed particle hydrodynamics (SPH) is a Lagrangian method that does not use a lattice, large deformations and movements can be easily analyzed, so it has the potential to be applied to such gas defect prediction as a methodology. However, numerical simulation of the twophase flow, in particular those with large density ratio, has been challenging. It is because the discontinuities of the density and the sharp pressure gradient exist over the interface. Authors have developed a two-phase flow SPH methodology that can be applied to the foundry filling process containing gas.

The weakly compressible SPH(WCSPH) was used as the analysis method. Strictly speaking, water has a slight compressibility. The weak compressible SPH is solved by using NS equations and equations of state (EOS) alternately. The Tait equation was used for EOS.

The methodology developed in this study is applied to typical problems of gas-liquid twophase flow including a dam break in air as well as a single bubble rising in water. The 2D as well as 3D bubble rising results show that both bubble shapes became similar to spheroids as shown by Grace. The water tests regarding the gas entrainment in the inclined part of the rectangular cavity and air exhaust from backstep part were also analized. Good agreement between the water tests and simulation results demonstrates the success of the SPH
methodology development.
The methodology in this study is applied to the 3D casting filling process simulation considering air entrainment.

## 2 TWO-PHASE FLOW SPH FORMULATION

### 2.1 Governing equations

The two-phase flow is considered in this study, and both phases obey the governing equations of a viscous Newtonian fluid.

$$
\begin{align*}
& \frac{D \rho}{D t}=-\rho \nabla \cdot \boldsymbol{u}  \tag{1}\\
& \frac{D u}{D t}=-\frac{1}{\rho} \nabla p+v \nabla^{2} \boldsymbol{u}+\boldsymbol{g}+\boldsymbol{f} \tag{2}
\end{align*}
$$

$\rho$ : density
$\boldsymbol{u}$ : velocity vector
$p$ : pressure
$v$ : kinematic viscosity coefficient
$\boldsymbol{g}$ : gravity
$f$ : surface tension
Water has a slight compressibility and is strictly weakly compressible. The weakly compressible SPH method solves using NS equations and equations of state alternately. The following equation of state is used throughout this paper.

$$
\begin{equation*}
p=B\left[\left(\frac{\rho}{\rho_{0}}\right)^{\gamma}-1\right] \tag{3}
\end{equation*}
$$

Where, $\rho_{0}$ is reference density and B is as follows:

$$
\begin{equation*}
B=\frac{\rho c_{s}^{2}}{\gamma} \tag{4}
\end{equation*}
$$

For water or liquid metals we use $\gamma=7$, and for gas $\gamma=1$ respectively.
The formulation used here is weakly compressible and is near the incompressible limit by choosing a sound speed $c_{s}$. Consequently the sound speed $c_{s}$, in the analysis, should be set about 10 times the maximum fluid velocity appearing in the analysis, not a physical value.

### 2.2 SPH formulation

The momentum equation (2) without gravity is discretized as follows;

$$
\begin{equation*}
\frac{D u_{i}}{D t}=\frac{1}{m_{i}} \sum_{j}\left(V_{i}^{2}+V_{j}^{2}\right)\left[-p_{i j} \frac{\partial W}{\partial r_{i j}} e_{i j}+\mu_{i j} \frac{u_{i j}}{r_{i j}} \frac{\partial W}{\partial r_{i j}}\right]+f_{i} \tag{5}
\end{equation*}
$$

Where

$$
\begin{align*}
& r_{i j}=\left|r_{i}-r_{j}\right|  \tag{6}\\
& V_{i}=\frac{m_{i}}{\rho_{i}} \tag{7}
\end{align*}
$$

Special treatment is needed for the interfaces of different phases. The density $\rho$ is calculated by the following equation.

$$
\begin{equation*}
\rho_{i}=m_{i} \sum_{j} W_{i j} \tag{8}
\end{equation*}
$$

Where $m, W$ are mass and Kernel function respectively.
The pressure and viscous terms are approximated as follows[1]:

$$
\begin{align*}
p_{i j} & =\frac{\left(\rho_{j} p_{i}+\rho_{i} p_{j}\right)}{\left(\rho_{i}+\rho_{j}\right)}  \tag{9}\\
\mu_{i j} & =\frac{2 \mu_{i} \mu_{j}}{\left(\mu_{i}+\mu_{j}\right)} \tag{10}
\end{align*}
$$

The following equation[2] is used for the surface tension term $f$.

$$
\begin{align*}
& f_{i}=-\frac{\sigma_{i}}{\rho_{i}} \kappa_{i} \frac{d c_{i}}{d x}  \tag{11}\\
& \frac{d c_{i}}{d x}=\frac{1}{V_{i}} \sum_{j}\left(V_{i}^{2}+V_{j}^{2}\right) c_{i j} \frac{\partial W}{\partial r_{i j}} e_{i j}  \tag{12}\\
& \kappa_{i}=d \frac{\sum_{j}\left(n_{i}-\varphi_{i j} n_{j}\right) e_{i j} \frac{\partial W}{\partial r_{i j}} V_{j}}{\sum_{j} r_{i j} \frac{\partial W}{\partial r_{i j}} V_{j}} \tag{13}
\end{align*}
$$

Where d is the spatial dimension and $\varphi_{i j}$ is defined as follows:
$\varphi_{i}{ }^{j}=\begin{aligned} & -1 \text { if particle } i \text { and } j \text { belongs to different phase } \\ & +1 \text { if particle } i \text { and } j \text { belongs to same phase }\end{aligned}$

## 3 NUMERICAL RESULTS

### 3.1 2D Analysis of Dam Break in Air

The water column collapse was analyzed by changing the gas density. The calculation conditions are shown in Figure 3.1. The aspect ratio (H/L) of the water column was set to 2 . The density ratios of gas and water were changed parametrically to $1 / 1000,1 / 100$ and $1 / 10$.


Figure 3.1 Calculation conditions for water column collapse in gas


Figure 3.2 Calculation results of water column collapse in gas

Figure 3.2 shows the calculation results for the density ratios of $1 / 1000$ and $1 / 10$.
Here, $t^{*}=t \sqrt{(2 g / L)}$ The gas around the water creates a vortex above the collapsing water column as shown in Figure 3.3.


Figure 3.3 Vector representation of flow in gas water column collapse at $\mathrm{t}^{*}=3.13$


Figure 3.4 Changes in the tip position in the calculation of water column collapse in gas
Figure 3.4 shows the time change of the tip position of the collapsed water column. Experimental data is also shown. The calculation result without considering the gas has a slightly faster tip speed than the experimental data. The calculation result of the density ratio $1 / 1000$ is also slightly faster than the experimental data. When the density ratio is increased, the change in tip speed is small in the case of a density ratio of $1 / 100$, but when it becomes about $1 / 10$, the tip speed clearly slows down.

### 3.2 2D Bubble Rising Analyses

Analysis of a bubble rising in water is often taken up as a typical subject of gas-liquid twophase flow. Figure 3.5 shows the analysis system for a bubble rising in the water. The bubble
/ water density ratio is $1 / 1000$. When a single bubble rises in water by buoyancy, surface tension, viscosity, and inertial force determine the bubble behavior. Since the experimental results of this bubble behavior are summarized by Grace[3], the analysis conditions are plotted in Grace's diagram as shown in Figure 3.6. Mo, Bo and Re are defined as follow:

$$
\begin{align*}
M o & =g \mu^{4} / \rho \sigma^{3}  \tag{15}\\
B o & =g d^{2} \rho / \sigma  \tag{16}\\
R e & =\left(B o^{3} / M o\right)^{\frac{1}{4}} \tag{17}
\end{align*}
$$



Case 1
$\mathrm{Mo}=2.58 \times 10^{-11}$
$\mathrm{Bo}=212.8$
$\mathrm{Re}=24800$
$\mathrm{R}=20 \mathrm{~mm}$

Case 2
$\mathrm{Mo}=2.58 \times 10^{-1}$
$\mathrm{Bo}=53.2$
$\mathrm{Re}=8766$
$\mathrm{R}=10 \mathrm{~mm}$

Figure 3.5 Analysis system of bubbles rising in water


Figure 3.6 Explanation of analysis case by Grace's experimental data
According to the figure, it is expected that the case 1 will be a spherical cap and the case 2 will be a spheroid. The calculation result of Case 1 is shown in Figure 3.7. Here, the dimensionless time $\mathrm{t} *$ is $t^{*}=t \sqrt{(g / R)}$, where R is bubble radius. The bubble shape is close to the spherical cap as epected. The calculation result of Case 2 is shown in Figure 3.8.

The shape of the bubble is similar to that of a spheroid as expected by the Grace chart.


Figure 3.7 Analysis result of case 1


Figure 3.8 Analysis result of case 2
The bubble rising speed (terminal velocity) for cases 1 and 2 was almost the same as the experimental result [4] shown in Figure 3.9.


Figure 3.9 Results on the rate of bubble rise

### 3.3 2D Analyses of Gas Entrainment in Inclined Portion of Rectangular Cavity

A water test of vertical push-up from the bottom into a rectangular cavity on a 10 mm thick acrylic flat plate has been conducted[5]. In the experimental device (Figure 3.10 left), the rotary motion of the motor is changed to a vertical motion with a ball screw, and the piston is
moved at a constant speed to push the water in the cylinder into the cavity. The initial water level is set 50 mm below the bottom of the cavity.
The push-up speed is 1.0 and $2.0 \mathrm{~m} / \mathrm{sec}$. In the initial state, the cavity is filled with air at room temperature. Two vents are arranged at the top of the cavity. The filling of water in the cavity is video-recorded using a CCD camera. When the gate speed is $1.0 \mathrm{~m} / \mathrm{sec}$, the water that has passed through the gate will soon spread to the left and right, entraining gas at the bottom of the cavity and colliding with the side surface. After that, the filling proceeded while the central part of the cavity swelled. On the other hand, when the gate speed was $2.0 \mathrm{~m} / \mathrm{sec}$, the inflowing water rose vigorously while the tip part swelled and collided with the upper surface of the cavity. Entrained a large amount of gas. These tests were modeled as shown in Figure 3.10 and analyzed by the developed SPH method. The density ratio of gas to water is $1.2 / 1000$. The distribution of flow velocity at the gate part seems to be the fastest in the center and the slowest in the part near the wall in the experiment, however in the analysis, the condition of constant flow velocity was given to the inlet part.


Figure 3.10 Experimental apparatus fof the vertical squeeze type casting and its analysis model
Figure 3.11 shows a comparison of the analysis and test results during the filling process for a push-up speed of $1.0 \mathrm{~m} / \mathrm{sec}$. The water gradually spreads laterally and rises to proceed with filling. Air remains in the channel expansion part (inclined part), and it can be seen that air is entrained at 0.16 sec in the test and at 0.24 sec in the analysis. The free surface shape is raised on both side walls, and the analysis well simulates the test results.

Figure 3.12 shows a comparison of the analysis and test results during the filling process for a push-up speed of $2.0 \mathrm{~m} / \mathrm{sec}$. The water rises slightly laterally and collides directly with the upper part of the cavity, and then fills the lower part along the outer circumference of the cavity. After the water collides with the upper cavity, the vent is closed. Therefore, no gas is emitted[5]. The analysis generally expresses the flow of the experiment. The analysis ended abnormally after 0.18 sec , because the vent was closed and the air particles were not discharged and consequently the number density of air particles increased abnormally.


Figure 3.11 comparison of the analysis and test result for case 1


Figure 3.12 Comparison of the analysis and test result for case 2

### 3.4 2D Analysis of Air Exhaust from Backward Step Part of Die

A water model experiment has been conducted on the air exhaust behavior in the backward step part [6]. The experimental equipment (Figure 3.13 left) consists of a water tank (1), a direct acting 2-port valve (2) and a test unit (3). The water in the tank pressurized by the air pressure from the compressor flows into the test section by opening the solenoid valve. The
test section (Figure 3.13, right) has a 20 mm thick transparent acrylic plate with a 2.0 mm deep groove and a flow path configuration that rapidly expands from a width of 15 mm to a width of 30 mm . A film sheet with a thickness of 0.02 mm is attached to the part other than the flow path. It has been reported over 7 cases of flow velocity, but since there is a photograph of the flow condition in the case of $u=8.41 \mathrm{~m} / \mathrm{sec}$, this was analyzed. The velocity $u$ is defined as a value obtained by dividing the water discharge amount by the crosssectional area of the width $15 \mathrm{~mm} \times$ the thickness 2.02 mm at the entrance of the test section. If the corresponding Re is defined by the step of 15 mm in the enlarged part, $\mathrm{Re}=1.26 \times 10^{5}$.


Figure 3.13 experimental equipment
Comparison of analysis results and experimental results [6] is shown in Figure 3.14 for each time. Here, the origin of time is the moment when the tip of the water flow enters the channel expansion portion. The analysis does not include the turbulent effect. At 12 ms in Figure (B), it collides with the downstream wall surface and flows to the right along the downstream wall surface. In Figure (C), it contacts the corner on the downstream right side of the channel expansion part at 22 ms in the experiment and 25 ms in the analysis. According to the literature, it is stated that "after that, a flowing upstream direction occurs along the enlarged flow path ", but in the analysis, the flowing upstream direction is extremely small. In Figure (D), there is a description that "a vortex containing bubbles is formed in the enlarged flow path". In the analysis, since the flowing upstream is small, only the vortex on the air side is conspicuous. In the literature, "then, as shown in Figures (E) and (F), the bubbles contained in the enlarged flow path are ejected into small bubbles due to the disturbance of the gasliquid interface and gradually discharged. Most of the air is trapped in the enlarged flow path." Most of the air remains trapped in the enlarged part by the analysis too.

(A) 4 ms

(B) 12 ms

(C)

(D) 38 ms

25ms(Analysis)
22 ms (test)


Figure 3.14 Comparison of analysis results and experimental results

### 3.5 3D Bubble Rising Analysis

A three-dimensional analysis of a bubble rising in water was performed. Figure 3.15 shows the comparison between two-dimensional analysis model and three dimensional one. Since 3D analysis requires a lot of calculation time, the analysis area is limited compared to 2D. The bubble / water density ratio was $1 / 1000$. Figure 3.16 shows the calculation results for the bubble radius $\mathrm{R}=10 \mathrm{~mm}$ together with the two-dimensional results. Here, $t^{*}=t \sqrt{(g / R)}$. The bubble shape at the time of rising became similar to a spheroid as shown in the experimental result summarized by Grace[3].


Figure 3.15 3D model and 2D model


Figure 3.16 Calculation results for the bubble radius $\mathrm{R}=10 \mathrm{~mm}$

## 4 ANALYSIS OF GAS ENTRAINMENT IN DIECAST

The mold filling process considering air entrainment in the die cast are simulated using the two-phase flow SPH method described in chapter 2[7].

### 4.1 Analysis Model

The particle model that simplifies the 3D die-casting shape in the field is used in this simulation is shown in Figure 4.1.


Figure 4.1 Analysis model for die casting
Total number of particles is 226,600, including initial air particles of 135,500 . The particle length is 0.1 cm . And particles with a specified speed flow in continuously at the molten metal inflow gate is shown in Figure 4.1 left.

### 4.2 Analysis Model

The material properties used in this simulation is shown in Table 4.1. The initial velocity of liquid Al alloy is $25 \mathrm{~m} / \mathrm{sec}$ and keeps constant during filling process.

Table4.1 Material Properties used for Analysis of Gas Entrainment in Diecast

| Die-Cast | $\mathrm{L}=21.8 \mathrm{~cm}, \mathrm{~W}=9.9 \mathrm{~cm}, \mathrm{H}=6.2 \mathrm{~cm}, \mathrm{t}=0.2 \mathrm{~cm}$ |  |
| :--- | :--- | :--- |
| Gas | Density | $1.293 \mathrm{Kg} / \mathrm{m}^{3}$ |
|  | Kinematic Viscosity Coefficient | $1.38 \times 10^{5} \mathrm{~m}^{2} / \mathrm{s}$ |
|  | Initial Pressure | 1.013 Pa |
|  | Density | $2500 \mathrm{Kg} / \mathrm{m}^{3}$ |
|  | Kinematic Viscousity Coefficient | $1.0 \times 10^{6} \mathrm{~m}^{2} / \mathrm{s}$ |
|  | Surface Tension | $0.9 \mathrm{~N} / \mathrm{m}$ |
|  | Ingate Velocity | $25 \mathrm{~m} / \mathrm{s}$ |

### 4.3 Results

Figure 4.2 shows the results of the two-phase flow SPH simulation. it can be confirmed that from the start of injection, the liquid Al alloy splits to the left and right in the Die-Cast, the eight entrances pass through the narrow gate and flow toward the vent part and the several airs remain after the complete filling. The maximum flow velocity occurred at the gate.


Figure 4.2 Analysis results of gas entraiment during die casting

## 5 CONCLUSIONS

- The gas-liquid two-phase flow was formulated using SPH.
- Authors analyzed several two-phase phenomina including the gas entrainment. All the analysis results are consistent with or similar to the test results.
- The developed methodology was applied to gas entrainment during the die cast filling process.


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