Explicit parametric solutions of lattice structures with Proper Generalized Decomposition (PGD)

Applications to the design of 3D-printed architected materials

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Abstract Architected materials (or metamaterials) are constituted by a unit-cell with a complex structural design repeated periodically forming a bulk material with emergent mechanical properties. One may obtain specific macro-scale (or bulk) properties in the resulting architected material by properly designing the unit-cell. Typically, this is stated as an optimal design problem in which the parameters describing the shape and mechanical properties of the unit-cell are selected in order to produce the desired bulk characteristics. This is especially pertinent due to the ease manufacturing of these complex structures with 3D printers. The Proper Generalized Decomposition (PGD) provides explicit parametric solutions of parametric PDEs. Here, the same ideas are used to obtain parametric solutions of the algebraic equations arising from lattice structural models. Once the explicit parametric solution is available, the optimal design problem is a simple post-process. The same strategy is applied in the numerical illustrations, first to a unit-cell (and then homogenized with periodicity conditions), and in a second phase to the complete structure of a lattice material specimen.

Keywords Parametric lattice structure · Proper Generalized Decomposition · Explicit parametric solution · architected materials · 3D printing

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1 Introduction

Architected materials (or metamaterials) are built as tessellations of small-scale structures (also named as cells). The cells are designed to obtain specific physical properties of the bulk. This concerns often the mechanical response, which is the scope of this work, but not only. The same concept is also used to obtain astonishing thermic, acoustic, optic or electromagnetic properties, see for instance [5]. Thus, as stated in [26], scientific and technical communities trend toward obtaining novel macro-structural material responses via micro-structural design. Achieving these new material properties is expected to highly impact the design of new devices in many fields of science and technology, see [8,10]. Just to mention a few applications, the use of architected materials results in negative or null thermal expansion coefficients [30,15], pentamodes or *simile-fluid* solids [21,17,2], elastic buckling or snapping based metamaterials [25,23], negative index of optical refraction [16,13] and materials with negative Poisson's ratio [7,14,24], also known as *auxetic* materials.

A natural problem arising in this context is the design of an architected material with tailored properties. This results in an *inverse problem* consisting in identifying the design parameters of the cell producing the desired properties in the bulk material. This is already addressed by the early works of Sigmund [28,29] in the context of computational mechanics with homogenization. The main difficulty of handling a parametric description of the micro-structure and to determine its influence in the emerging properties of the bulk is the multidimensional character of the problem. The computational complexity increases exponentially with the number of parameters and hence the burden of the inverse problem. Quoting [26], this problem "[...] is generally ill-posed, and making progresses requires a combination of computational strategies to strengthen our physical intuition".

A particular and interesting application of these ideas is producing auxetic metamaterials with additive manufacturing (or 3D printing). The seed cell is in this case a 3D printed lattice material with parametrically described shape and size. In this context, different numerical models testing diverse scenarios for the micro-structure have successfully predicted the material response of the bulk. For instance, taking an inverse honeycomb configuration, first proposed by [1] and extensively analyzed later by [27,12,18]. However, as pointed out above, the computational burden is still a bottleneck when the number of parameters is moderately large.

One possible solution to affordably deal with the so-called *curse of dimensionality* is using Reduced Order Models (ROM). Here, we focus on a particular ROM: the Proper Generalized Decomposition (PGD), presenting a separable approximation computed using a combination of a greedy algorithm (to compute successively the terms) and an alternated directions scheme (to compute iteratively the modes in each term), see [9,32]. This technique is specially well suited to simulate the behavior of the parametric structure (with parameters describing both geometrical and material properties of the cell) because it provides an explicit parametric solution, also denoted as *computa-*

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tional vademecum. In the PGD philosophy, the computational vademecum is obtained *offline*, often using High Performance Computing resources. Then, to solve the inverse problem, the parametric design space is browsed as a post-process, in an *online* phase providing real-time responses.

This paper aims at presenting a new PGD formulation to efficiently deal with the parametric lattice material problem. The explicit parametric solution (computational vademecum) is to be exploited to derive the macro-scale properties of the resulting architected material and hence to efficiently solve the inverse problem under consideration.

The remainder of the paper is structured as follows. Section 2 describes the parametric discrete problem and the functional setup. Section 3 introduces in detail the generalization of the Proper Generalized Decomposition to the case of intrinsically discrete lattice structures. Then, in Section 4 the proposed strategy is applied to solve the parametric model in different structural configurations. Some concluding remarks are drawn in Section 5.

2 Problem statement: parametric lattice structure

The shape and material properties of a lattice structure are characterized by n_p parameters μ_i , for $i = 1, 2, \dots, n_p$, each ranging in a real interval $I_i \subset \mathbb{R}$. For the sake of shortening the notation, the n_p parameters are collected in a vector $\boldsymbol{\mu} = [\mu_1 \mu_2 \dots \mu_{n_p}]^T$. Accordingly, $\boldsymbol{\mu}$ ranges in the multidimensional parametric space $\mathcal{D} = I_1 \times I_2 \times \dots \times I_{n_p} \subset \mathbb{R}^{n_p}$.

The deformation of the structure is described by a vector of generalized displacements (nodal displacements and rotations), $\mathbf{U}(\boldsymbol{\mu})$, which is the unknown of the parametric problem. The number of degrees of freedom of the problem is denoted by n_{dof} and therefore for every value of the parameters, $\mathbf{U}(\boldsymbol{\mu}) \in \mathbb{R}^{n_{\text{dof}}}$.

The parametric input data boils down into the stiffness matrix $\mathbf{K}(\boldsymbol{\mu}) \in \mathbb{R}^{n_{\text{dof}} \times n_{\text{dof}}}$ and the vector of generalized nodal forces (including forces and moments), $\mathbf{F}(\boldsymbol{\mu}) \in \mathbb{R}^{n_{\text{dof}}}$. Thus, the equilibrium equation that has to be solved to obtain $\mathbf{U}(\boldsymbol{\mu})$ reads

$$\mathbf{K}(\boldsymbol{\mu}) \mathbf{U}(\boldsymbol{\mu}) = \mathbf{F}(\boldsymbol{\mu}). \quad (1)$$

Equation (1) is stated in a parametric fashion, in the sense that input data $\mathbf{K}$ and $\mathbf{F}$ depend on $\boldsymbol{\mu}$ and therefore also the solution $\mathbf{U}$ depends on $\boldsymbol{\mu}$. Thus, all these mathematical objects are fields taking values in the multidimensional parametric space $\mathcal{D}$. In the following it is assumed that the parametric dependence of the input data is regular enough to be square integrable, that is $\mathbf{F}(\boldsymbol{\mu}) \in [\mathcal{L}_2(\mathcal{D})]^{n_{\text{dof}}}$ and $\mathbf{K}(\boldsymbol{\mu}) \in [\mathcal{L}_2(\mathcal{D})]^{n_{\text{dof}} \times n_{\text{dof}}}$. Note that the parametric functional space $\mathcal{L}_2(\mathcal{D})$ is also expressed in terms of the sectional spaces $\mathcal{L}_2(I_i)$, $i = 1, 2, \dots, n_p$. Namely, $\mathcal{L}_2(\mathcal{D}) = \mathcal{L}_2(I_1) \otimes \mathcal{L}_2(I_2) \otimes \dots \otimes \mathcal{L}_2(I_{n_p})$ which means that freezing all the parameters but one, the remaining functional dependence is square integrable.

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Matrix $\mathbf{K}$ is build up assembling the contribution of all the beam elements of the lattice structure. The formulation used in the numerical examples correspond to Bernoulli beam elements. However, all the developments are valid for any beam or structural element formulation. The parameters included in $\boldsymbol{\mu}$ do characterize the geometry and material properties of the individual elements. The same concept is readily generalized to the parameterization of substructures that are also assembled into the global structure. This strategy is particularly useful in the context of architected materials, in which a unit-cell is periodically replicated in the bulk structure.

Equation (1) is also expressed in integral form. First, the residual of (1) is introduced as

$$\mathbf{R}(\mathbf{U}(\boldsymbol{\mu})) := \mathbf{F}(\boldsymbol{\mu}) - \mathbf{K}(\boldsymbol{\mu}) \mathbf{U}(\boldsymbol{\mu}). \quad (2)$$

Thus, using the weighted residuals idea, one can state that $\mathbf{U}(\boldsymbol{\mu})$ is the solution of (1) if and only if

$$\int_{I_1} \int_{I_2} \dots \int_{I_{n_p}} \delta \mathbf{U}(\boldsymbol{\mu})^\top \mathbf{R}(\mathbf{U}(\boldsymbol{\mu})) d\mu_{n_p} \dots d\mu_2 d\mu_1 = 0 \quad (3)$$

for all $\delta \mathbf{U}(\boldsymbol{\mu}) \in [\mathcal{L}_2(\mathcal{D})]^{n_{\text{dof}}}$.

Note that no integration is performed in the physical space. The integrals involve only the parametric space $\mathcal{D}$. This is due to the algebraic nature of (1), which can be seen as already discretized in space. The role of the space integration (energy product) is played here by the scalar product of the residual (forces) and the test function (virtual displacements).

The problem represented by the equivalent equations (1) and (3) is discretized in order to devise a numerical solver. Finite-dimensional spaces $V_i \subset \mathcal{L}_2(I_i)$, $i = 1, 2, \dots, n_p$, of dimension $n_{d,i}$ are introduced to approximate each sectional parametric space. Accordingly, the space where the discrete approximation to $\mathbf{U}(\boldsymbol{\mu})$ lies is $[V_1 \otimes V_2 \otimes \dots \otimes V_{n_p}]^{n_{\text{dof}}}$ with overall dimension equal to

$$n_{\text{Full}} = n_{\text{dof}} \prod_{i=1}^{n_p} n_{d,i}. \quad (4)$$

The total number of unknowns n_{Full} is growing fast with the number of parameters n_p , producing the so-called *curse of dimensionality*. In other words, the number of degrees of freedom in the full multidimensional problem, n_{Full} , grows exponentially with n_p .

Reduced Order Models (ROM) are possible alternatives to overcome this difficulty. In particular, the Proper Generalized Decomposition (PGD) is a ROM based on the idea of providing a separable decomposition of the multidimensional function and therefore reducing the exponential computational complexity into a linear one. Next section discusses the application of PGD to the structural problem under consideration.

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3 Proper Generalized Decomposition solver

3.1 Separable approximations

The unknown $\mathbf{U}(\boldsymbol{\mu})$ is to be approximated by a separable approximation $\mathbf{U}_{\text{PGD}}^n(\boldsymbol{\mu})$ with n terms, namely

$$\begin{aligned}\mathbf{U}(\boldsymbol{\mu}) &\approx \mathbf{U}_{\text{PGD}}^n(\boldsymbol{\mu}) = \sum_{m=1}^n \mathbf{u}^m G_1^m(\mu_1) G_2^m(\mu_2) \dots G_{n_p}^m(\mu_{n_p}) \\ &= \sum_{m=1}^n \mathbf{u}^m \prod_{i=1}^{n_p} G_i^m(\mu_i) \\ &= \mathbf{U}_{\text{PGD}}^{n-1}(\boldsymbol{\mu}) + \mathbf{u}^n G_1^n(\mu_1) G_2^n(\mu_2) \dots G_{n_p}^n(\mu_{n_p})\end{aligned}\quad (5)$$

where each term, for $m = 1, 2, \dots, n$ is determined by a displacement vector $\mathbf{u}^m$ describing the spatial mode and a set of parametric functions $G_i^m(\mu_i)$, for $i = 1, 2, \dots, n_p$. In the following, in order to alleviate the notation and where there is no ambiguity, the explicit dependence on μ_i is omitted and $G_i^m(\mu_i)$ is written as G_i^m .

In order to use PGD, the input data, $\mathbf{K}(\boldsymbol{\mu})$ and $\mathbf{F}(\boldsymbol{\mu})$, has to be expressed in separated form, that is

$$\mathbf{K}(\boldsymbol{\mu}) = \sum_{k=1}^{n_k} \mathbf{K}^k \prod_{i=1}^{n_p} B_i^k(\mu_i) \text{ and} \quad (6)$$

$$\mathbf{F}(\boldsymbol{\mu}) = \sum_{\ell=1}^{n_f} \mathbf{f}^\ell \prod_{i=1}^{n_p} S_i^\ell(\mu_i), \quad (7)$$

n_k and n_f being the number of terms needed to express $\mathbf{K}$ and $\mathbf{F}$ in a separable manner, $\mathbf{K}^k$ and $\mathbf{f}^\ell$ the corresponding spatial modes and $B_i^k(\mu_i)$ and $S_i^\ell(\mu_i)$ the parametric functions. For the sake of simplifying the notation, also the parameter dependence of B_i^k and S_i^ℓ is omitted in the following. In the case the input data are not originally separated, a separable approximation has to be obtained as a pre-process; see [31] for a discussion on the possible errors introduced in this phase.

3.2 Sectional norms

The PGD solver is based on the discretization and solution of the integral form of the problem presented in (3) with separable approximations. Formulating the PGD requires introducing sectional norms, that is norms along each of the independent parametric dimensions that allow measuring the modes.

For instance, the modes $G_i^m \in \mathcal{L}_2(I_i)$ describing $\mathbf{U}_{\text{PGD}}$ in (5) are naturally measured with the standard norm in $\mathcal{L}_2(I_i)$, namely

$$\|G_i^m\|^2 = \int_{I_i} (G_i^m)^2 d\mu_i. \quad (8)$$

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The choice for the sectional norm affecting the space (or physical) dimension, that is the norm to measure the modal vectors $\mathbf{u}^m \in \mathbb{R}^{\text{n}_{\text{dof}}}$, is not as trivial. The obvious choice of selecting the Euclidean scalar product in $\mathbb{R}^{\text{n}_{\text{dof}}}$ would yield

$$\|\mathbf{u}^m\|^2 = [\mathbf{u}^m]^\top \mathbf{u}^m.$$

The Euclidean norm, however, does not take into account the nature of the generalized nodal displacements (mixing displacements and rotations) and the resulting measure lacks of physical meaning. A more suitable choice is using a structural mass matrix $\mathbf{M}_u$ with sound physical rationale that results in a norm such that

$$\|\mathbf{u}^m\|^2 = [\mathbf{u}^m]^\top \mathbf{M}_u \mathbf{u}^m. \quad (9)$$

Once these norms are available, it is interesting normalizing the modes, that is taking

$$\tilde{\mathbf{u}}^m = \frac{1}{\|\mathbf{u}^m\|} \mathbf{u}^m \text{ and } \tilde{G}_i^m = \frac{1}{\|G_i^m\|} G_i^m. \quad (10)$$

Thus, the separated representation of (5) is rewritten as

$$\mathbf{U}_{\text{PGD}}^n(\boldsymbol{\mu}) = \sum_{m=1}^n \beta^m \tilde{\mathbf{u}}^m \prod_{i=1}^{\text{n}_p} \tilde{G}_i^m, \quad (11)$$

being $\beta^m = \|\mathbf{u}^m\| \prod_{i=1}^{\text{n}_p} \|G_i^m\|$ the amplitude of term m of the separable sum.

Note that β^m provides key information on the relative importance of the different modes in the separable approximation and therefore it is used as one of the criteria to decide whether the number of terms, n , is sufficient to obtain the desired accuracy.

A global norm in $[\mathcal{L}_2(\mathcal{D})]^{\text{n}_{\text{dof}}}$ is also introduced such that,

$$\|\mathbf{U}(\boldsymbol{\mu})\|_{\text{glob}}^2 = \int_{I_1} \int_{I_2} \dots \int_{I_{\text{n}_p}} \mathbf{U}(\boldsymbol{\mu})^\top \mathbf{M}_u \mathbf{U}(\boldsymbol{\mu}) d\mu_{\text{n}_p} \dots d\mu_2 d\mu_1 \quad (12)$$

for any $\mathbf{U}(\boldsymbol{\mu}) \in [\mathcal{L}_2(\mathcal{D})]^{\text{n}_{\text{dof}}}$

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3.3 Greedy strategy and rank-one approximation

The PGD methodology aims at numerically solving (3) using an approximation of the form shown in (5). This is performed using a greedy strategy, that is computing sequentially the terms of the sum in (5). Thus, we start computing $\mathbf{U}_{\text{PGD}}^1$, and then, when $\mathbf{U}_{\text{PGD}}^1$ is available, compute $\mathbf{U}_{\text{PGD}}^2$, and then $\mathbf{U}_{\text{PGD}}^3$... Each of the steps of the greedy algorithm (that is, updating some $\mathbf{U}_{\text{PGD}}^{n-1}$ into $\mathbf{U}_{\text{PGD}}^n$) consists in solving a rank-one approximation problem.

Accordingly, it is assumed that $\mathbf{U}_{\text{PGD}}^{n-1}$ is known and that

$$\mathbf{U}_{\text{PGD}}^n = \mathbf{U}_{\text{PGD}}^{n-1} + \mathbf{u} \prod_{i=1}^{\text{n}_p} G_i, \quad (13)$$

where the superscript n in $\mathbf{u}^n$ and G_i^n is here omitted to simplify the notation.

The problem consists now in finding $\mathbf{u}$, $G_1, G_2, \dots, G_{n_p}$ such that $\mathbf{U}_{\text{PGD}}^n$ fulfills (3). The complete unknown function $\mathbf{u} \prod_{i=1}^{n_p} G_i$ is said to be of rank one because it is build as the product of sectional functions. The problem at hand is nonlinear (because the unknown functions are multiplying each other) but with a number of degrees of freedom much lower than the original one. Recall that the discrete version of the full linear problem has a number of degrees of freedom n_{Full} given in (4). Instead, with the PGD strategy, each rank-one approximation problem is nonlinear but with a number of degrees of freedom, n_{RankOne} given by:

$$n_{\text{RankOne}} = n_{\text{dof}} + \sum_{i=1}^{n_p} n_{d,i}. \quad (14)$$

Typically, due to the additive nature of n_{RankOne} in terms of the sectional dimensions, $n_{\text{RankOne}} \ll n_{\text{Full}}$ holds and therefore the reduction of the dimension of the problem is well worth the difficulties associated with the newly acquired nonlinear character.

Due to the incremental character of (13), the expression of the residual (2) is rewritten as

$$\mathbf{R}(\mathbf{U}_{\text{PGD}}^n) = \mathbf{R}(\mathbf{U}_{\text{PGD}}^{n-1}) - \mathbf{K}(\boldsymbol{\mu}) \mathbf{u} \prod_{i=1}^{n_p} G_i, \quad (15)$$

which, using (6), becomes

$$\begin{aligned} \mathbf{R}(\mathbf{U}_{\text{PGD}}^n) &= \mathbf{R}(\mathbf{U}_{\text{PGD}}^{n-1}) - \left[\sum_{k=1}^{n_g} \mathbf{K}^k \prod_{i=1}^{n_p} B_i^k \right] \mathbf{u} \prod_{i=1}^{n_p} G_i \\ &= \mathbf{R}(\mathbf{U}_{\text{PGD}}^{n-1}) - \sum_{k=1}^{n_g} \left[\prod_{i=1}^{n_p} B_i^k G_i \right] \mathbf{K}^k \mathbf{u}. \end{aligned} \quad (16)$$

The test function $\delta \mathbf{U}$ in (3) (parametric virtual displacement, the explicit dependence on $\boldsymbol{\mu}$ is omitted now in the notation) has to be selected now in accordance with the unknown of the rank-one problem consisting in introducing in (3) the residual as defined in (15) or (16). This test function is taken as a variation of the actual unknown $\mathbf{u} \prod_{i=1}^{n_p} G_i$ and yields

$$\delta \mathbf{U} = \delta \mathbf{u} \prod_{i=1}^{n_p} G_i + \mathbf{u} \sum_{i=1}^{n_p} \delta G_i \prod_{\substack{j=1 \\ j \neq i}}^{n_p} G_j. \quad (17)$$

That is, instead of taking an arbitrary $\delta \mathbf{U} \in [\mathcal{L}_2(\mathcal{D})]^{n_{\text{dof}}}$ as in (3), one must take arbitrary $\delta \mathbf{u} \in \mathbb{R}^{n_{\text{dof}}}$ and $\delta G_i \in \mathcal{L}_2(I_i)$, for $i = 1, 2, \dots, n_p$. Then, in the numerical version, the finite-dimensional sectional spaces V_i replace $\mathcal{L}_2(I_i)$.

The numerical strategy to deal with the nonlinearity of this problem is based on the alternated directions idea [3].

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3.4 Alternated directions solver

The alternated directions strategy to solve the rank-one nonlinear problem consists in successively solving for each one of the unknowns assuming that the rest of them are known.

3.4.1 Computing $\mathbf{u}$

Thus, the first step consists in, assuming G_i known, for $i = 1, 2, \dots, n_p$, compute $\mathbf{u}$ such that $\mathbf{R}(\mathbf{U}_{\text{PGD}}^n)$ as defined in (16) satisfies (3). Since G_i is known, δG_i is taken to be zero, for $i = 1, 2, \dots, n_p$, and $\delta \mathbf{U}$ in (17) reduces to

$$\delta \mathbf{U} = \delta \mathbf{u} \prod_{i=1}^{n_p} G_i.$$

Thus, in this case (3) becomes

$$\int_{\times_{j=1}^{n_p} I_j} \delta \mathbf{u}^\top \mathbf{R}(\mathbf{U}_{\text{PGD}}^n) \prod_{i=1}^{n_p} G_i d\mu_j = 0$$

for all $\delta \mathbf{u} \in \mathbb{R}^{n_{\text{dof}}}$. Note that the notation has been shortened in a single multidimensional integral but using $\times_{j=1}^{n_p} I_j$ instead of $\mathcal{D}$ because this notation is going to be useful in the following. The previous equation fulfilled for all $\delta \mathbf{u} \in \mathbb{R}^{n_{\text{dof}}}$ is equivalent to the following algebraic equation in $\mathbb{R}^{n_{\text{dof}}}$:

$$\int_{\times_{j=1}^{n_p} I_j} \mathbf{R}(\mathbf{U}_{\text{PGD}}^n) \prod_{i=1}^{n_p} G_i d\mu_i = \mathbf{0}.$$

Substituting $\mathbf{R}(\mathbf{U}_{\text{PGD}}^n)$ by the expression (16) one gets

$$\int_{\times_{j=1}^{n_p} I_j} \left(\mathbf{R}(\mathbf{U}_{\text{PGD}}^{n-1}) - \sum_{k=1}^{n_k} \left[\prod_{i=1}^{n_p} B_i^k G_i \right] \mathbf{K}^k \mathbf{u} \right) \prod_{i=1}^{n_p} G_i d\mu_i = \mathbf{0}.$$

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Using the expressions for $\mathbf{U}(\mu)$, $\mathbf{K}(\mu)$ and $\mathbf{F}(\mu)$ in (5), (6) and (7), respectively, and the definition of the residual in (2) the resulting equation reads

$$\left[\sum_{k=1}^{n_k} \mathbf{K}^k \underbrace{\prod_{i=1}^{n_p} \int_{I_i} B_i^k (G_i)^2 d\mu_i}_{c^k} \right] \mathbf{u} = \sum_{\ell=1}^{n_\ell} \overbrace{\mathbf{f}^\ell \prod_{i=1}^{n_p} \int_{I_i} S_i^\ell G_i d\mu_i}^{\hat{c}^\ell} \quad (18)$$

$$- \sum_{m=1}^{n-1} \left[\sum_{k=1}^{n_k} \mathbf{K}^k \underbrace{\prod_{i=1}^{n_p} \int_{I_i} B_i^k G_i^m G_i d\mu_i}_{c^{k,m}} \right] \mathbf{u}^m.$$

That is, a linear system of equations for $\mathbf{u}$:

$$\left[\sum_{k=1}^{n_k} \mathbf{K}^k c^k \right] \mathbf{u} = \sum_{\ell=1}^{n_\ell} \mathbf{f}^\ell \hat{c}^\ell - \sum_{m=1}^{n-1} \left[\sum_{k=1}^{n_k} \mathbf{K}^k c^{k,m} \right] \mathbf{u}^m, \quad (19)$$

where scalars c^k , $\hat{c}^\ell$ and $c^{k,m}$ are computable terms defined by:

$$c^k := \prod_{i=1}^{n_p} \int_{I_i} B_i^k (G_i)^2 d\mu_i, \quad (20)$$

$$\hat{c}^\ell := \prod_{i=1}^{n_p} \int_{I_i} S_i^\ell G_i d\mu_i \text{ and} \quad (21)$$

$$c^{k,m} := \prod_{i=1}^{n_p} \int_{I_i} B_i^k G_i^m G_i d\mu_i. \quad (22)$$

3.4.2 Computing G_i , for $i = 1, 2, \dots, n_p$

The subsequent steps consist in computing one parametric mode, say G_i , assuming that $\mathbf{u}$ and the rest of the modes G_j , for $j = 1, 2, \dots, n_p$ and $j \neq i$, are known and therefore the corresponding variations δG_j and $\delta \mathbf{u}$ are taken to be zero. Thus, the variation $\delta \mathbf{U}$ in (17) becomes

$$\delta \mathbf{U} = \mathbf{u} \delta G_i \prod_{\substack{j=1 \\ j \neq i}}^{n_p} G_j.$$

Using this variation in (3) for $\delta \mathbf{U}$ one gets that

$$\int_{I_i} \delta G_i \left[\int_{\times_{j \neq i} I_j} \mathbf{u}^\top \mathbf{R}(\mathbf{U}_{\text{PGD}}^n) \prod_{j \neq i} G_j d\mu_j \right] d\mu_i = 0 \quad (23)$$

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must hold for all $\delta G_i \in \mathcal{L}_2(I_i)$. In order to ease the reading, the range of j is omitted, emphasizing only the fact that $j \neq i$. Note that (23) is an equation for the unknown function $G_i(\cdot)$ that appears explicitly in $\mathbf{R}(\mathbf{U}_{\text{PGD}}^n)$, see (15). Indeed, the integral equation (23) results in the following equation for $G_i(\cdot)$

$$\int_{\times_{j \neq i} I_j} \mathbf{u}^\top \left[\mathbf{R}(\mathbf{U}_{\text{PGD}}^{n-1}) - G_i \mathbf{K}(\boldsymbol{\mu}) \mathbf{u} \prod_{j \neq i} G_j \right] \prod_{j \neq i} G_j d\mu_j = 0. \quad (24)$$

That is

$$G_i \left[\int_{\times_{j \neq i} I_j} \mathbf{u}^\top \mathbf{K}(\boldsymbol{\mu}) \mathbf{u} \prod_{j \neq i} (G_j)^2 d\mu_j \right] = \int_{\times_{j \neq i} I_j} \mathbf{u}^\top \mathbf{R}(\mathbf{U}_{\text{PGD}}^{n-1}) \prod_{j \neq i} G_j d\mu_j, \quad (25)$$

where both the term in square brackets in the left-hand side and the right-hand side of (25) are computable functions in I_i . The expression for these functions is readily determined using the forms of the data given in (5), (6) and (7). The resulting expressions read

$$G_i(\cdot) = \frac{\sum_{\ell=1}^{n_\ell} (\mathbf{u}^\top \mathbf{f}^\ell) \hat{d}_i^\ell(\cdot) - \sum_{m=1}^{n-1} \sum_{k=1}^{n_k} (\mathbf{u}^\top \mathbf{K}^k \mathbf{u}^m) d_i^{k,m}(\cdot)}{\sum_{k=1}^{n_k} (\mathbf{u}^\top \mathbf{K}^k \mathbf{u}) d_i^k(\cdot)}, \quad (26)$$

where the computable functions $d_i^k(\cdot)$, $\hat{d}_i^\ell(\cdot)$ and $d_i^{k,m}(\cdot)$ taking values in I_i are defined by

$$d_i^k(\cdot) := \left(\prod_{\substack{j=1; \\ j \neq i}}^{n_p} \int_{I_j} B_j^k (G_j)^2 d\mu_j \right) B_i^k(\cdot), \quad (27)$$

$$\hat{d}_i^\ell(\cdot) := \left(\prod_{\substack{j=1; \\ j \neq i}}^{n_p} \int_{I_j} S_j^\ell G_j d\mu_j \right) S_i^\ell(\cdot) \text{ and} \quad (28)$$

$$d_i^{k,m}(\cdot) := \left(\prod_{\substack{j=1; \\ j \neq i}}^{n_p} \int_{I_j} B_j^k G_j^m d\mu_j \right) B_i^k(\cdot) G_i^m(\cdot). \quad (29)$$

3.4.3 Convergence control and stopping criteria

For each term of the PGD solution, that is for each value of n , the alternated directions iterations are expected to converge to the best rank-one approximation of $\mathbf{u}_{\text{PGD}} \otimes \mathbf{G}_{\text{PGD}}$, being $\mathbf{u}_{\text{PGD}}$ and $\mathbf{G}_{\text{PGD}}$ common to this iterative algorithm (see Appendix A) requires a stopping criterion to decide whether the current iteration is acceptable or not. The stopping criterion is based on the stationarity of the solution: the iteration is validated if the modification from the previous iteration is small enough. Thus, assume that $\mathbf{u}$ and G_i , for $i = 1, 2, \dots, n_p$ characterize the previous iteration and that after the alternated directions loop the new approximation is given by $\mathbf{u}^{\text{new}}$ and G_i^{new} , for $i = 1, 2, \dots, n_p$. The stopping criteria is expressed in terms of the difference between the two successive iterations, measured with the norms introduced in Section 3.2.

In particular, a typical convergence criterion is to continue iterating while

$$\left\| \mathbf{u}^{\text{new}} \prod_{i=1}^{n_p} G_i^{\text{new}} - \mathbf{u} \prod_{i=1}^{n_p} G_i \right\|_{\text{Glob}} > \eta_{\text{tol}} \left\| \mathbf{u}^{\text{new}} \prod_{i=1}^{n_p} G_i^{\text{new}} \right\|_{\text{Glob}}, \quad (30)$$

where $\|\cdot\|_{\text{Glob}}$ is introduced in (12) and η_{tol} is a user-prescribed tolerance.

The computation of these norms is simplified by using the normalized modes, $\tilde{\mathbf{u}}$ and $\tilde{G}_i$, introduced in (10) and the amplitude β . Namely,

$$\begin{aligned} \left\| \mathbf{u}^{\text{new}} \prod_{i=1}^{n_p} G_i^{\text{new}} - \mathbf{u} \prod_{i=1}^{n_p} G_i \right\|_{\text{Glob}}^2 &= \left\| \mathbf{u}^{\text{new}} \prod_{i=1}^{n_p} G_i^{\text{new}} \right\|_{\text{Glob}}^2 + \left\| \mathbf{u} \prod_{i=1}^{n_p} G_i \right\|_{\text{Glob}}^2 \\ &\quad - 2(\mathbf{u}^{\text{new}}, \mathbf{u}) \prod_{i=1}^{n_p} (G_i^{\text{new}}, G_i) \\ &= (\beta^{\text{new}})^2 + \beta^2 \\ &\quad - 2\beta^{\text{new}}\beta (\tilde{\mathbf{u}}^T \mathbf{M}_{\mathbf{u}} \tilde{\mathbf{u}}^{\text{new}}) \prod_{i=1}^{n_p} \int_{I_i} \tilde{G}_i^{\text{new}} \tilde{G}_i d\mu_i, \end{aligned} \quad (31)$$

where all the sectional scalar products are denoted in a unified fashion by the bilinear operator $(\cdot, \cdot)$. Note that if the new iteration coincides with the previous one, this expression vanishes because the amplitudes are equal and the scalar product of the identical and normalized modes is equal to one.

In Appendix A the algorithm of the alternated directions solver is detailed, see Algorithm 1.

3.5 PGD compression

The n terms of PGD solution $\mathbf{U}_{\text{PGD}}^n$ may contain redundant information. This is associated with the greedy strategy employed to compute the successive terms, with no enforcement of any *orthogonality* between the successive modes. Thus, the number of terms in an *optimal* separable approximation required to achieve the same level of accuracy as in $\mathbf{U}_{\text{PGD}}^n$ is often much lower than n . This can be checked a posteriori in the 2D case, where the Singular Value Decomposition (SVD) provides an optimal separation, with the least number of terms. For higher dimensions, the problem still holds but there is no optimal solution to compare with (the High Order SVD, or HO-SVD, is no longer optimal).

In order to mitigate the effect of this phenomenon, a common practice in the PGD codes is to implement the so-called PGD compression, see the appendix in [22] where it is referred also as HO-PGD. This compression consists in a least-squares projection of $\mathbf{U}_{\text{PGD}}^n$ into the same approximation space, computed with the very same PGD strategy, that is combining a greedy algorithm for the terms and an alternated directions scheme for the modes.

In the context of the parametric structural problem (1), the PGD compression is formulated as follows. Let $\mathbf{U}_{\text{PGD}}^n$ be the *raw* PGD solution with n terms. Instead of solving the original equation (1) or its integral counterpart (3), now the objective is obtaining $\mathbf{U}_{\text{com}}$ minimizing the least-squares functional $\mathcal{J}$ defined as

$$\mathcal{J}(\mathbf{U}_{\text{com}}) = \|\mathbf{U}_{\text{com}} - \mathbf{U}_{\text{PGD}}^n\|_{\text{Glob}}^2. \quad (32)$$

More precisely, the aim is at computing a separated approximation $\mathbf{U}_{\text{com}}^{\hat{n}}$ with $\hat{n}$ terms expressed as

$$\mathbf{U}_{\text{com}}^{\hat{n}} = \sum_{\hat{m}=1}^{\hat{n}} \hat{\mathbf{u}}^{\hat{m}} \prod_i^{\mathbf{n}_p} \hat{G}_i^{\hat{m}} \quad (33)$$

using a PGD approach to minimize (32) with the expectation of getting a sufficiently accurate approximation to $\mathbf{U}_{\text{PGD}}^n$ with $\hat{n} \ll n$.

Again, the main idea of the PGD strategy is formulating the problem of finding a rank-one approximation. Thus, let us briefly describe how to compute the first term of $\mathbf{U}_{\text{com}}^{\hat{n}}$ taking

$$\mathbf{U}_{\text{com}} = \hat{\mathbf{u}} \prod_i^{\mathbf{n}_p} \hat{G}_i.$$

Following the expansion in (31), the expression for $\mathcal{J}(\mathbf{U}_{\text{com}})$ is reduced to its dependence on the unknowns $\hat{\mathbf{u}}, \hat{G}_1, \dots, \hat{G}_{\mathbf{n}_p}$ and reads

$$\begin{aligned} \mathcal{J}(\hat{\mathbf{u}}, \hat{G}_1, \dots, \hat{G}_{\mathbf{n}_p}) &= \left\| \hat{\mathbf{u}} \prod_{i=1}^{\mathbf{n}_p} \hat{G}_i \right\|_{\text{Glob}}^2 + \left\| \sum_{m=1}^n \mathbf{u}^m \prod_{i=1}^{\mathbf{n}_p} G_i^m \right\|_{\text{Glob}}^2 \\ &\quad - 2 \sum_{m=1}^n (\hat{\mathbf{u}}, \mathbf{u}^m) \prod_{i=1}^{\mathbf{n}_p} (\hat{G}_i, G_i^m). \end{aligned}$$

Again the alternated direction scheme is used here. The first step assumes that $\hat{G}_1, \dots, \hat{G}_{\mathbf{n}_p}$ are known to compute $\hat{\mathbf{u}}$. Note that the previous functional is quadratic for $\hat{\mathbf{u}}$ and therefore the sectional minimization problem results in solving a linear system of equations for $\hat{\mathbf{u}}$. Indeed, for given values of $\hat{G}_1, \dots, \hat{G}_{\mathbf{n}_p}$, the functional dependence on the remaining unknown reads

$$\begin{aligned} \mathcal{J}(\hat{\mathbf{u}}) &= (\hat{\mathbf{u}}^T \mathbf{M}_{\mathbf{u}} \hat{\mathbf{u}}) \overbrace{\prod_{i=1}^{\mathbf{n}_p} (\hat{G}_i, \hat{G}_i)}^{\lambda} + \left\| \sum_{m=1}^n \mathbf{u}^m \prod_{i=1}^{\mathbf{n}_p} G_i^m \right\|_{\text{Glob}}^2 \\ &\quad - 2 \sum_{m=1}^n (\hat{\mathbf{u}}^T \mathbf{M}_{\mathbf{u}} \mathbf{u}^m) \underbrace{\prod_{i=1}^{\mathbf{n}_p} (\hat{G}_i, G_i^m)}_{\gamma^m} \end{aligned}$$

and minimizing $\mathcal{J}(\hat{\mathbf{u}})$ requires solving $\frac{\partial \mathcal{J}(\hat{\mathbf{u}})}{\partial \hat{\mathbf{u}}} = 0$, that is

$$\mathbf{M}_{\mathbf{u}} \hat{\mathbf{u}} = \frac{1}{\lambda} \sum_{m=1}^n \gamma^m \mathbf{M}_{\mathbf{u}} \mathbf{u}^m \implies \hat{\mathbf{u}} = \frac{1}{\lambda} \sum_{m=1}^n \gamma^m \mathbf{u}^m. \quad (34)$$

Note that matrix $\mathbf{M}_{\mathbf{u}}$ cancels in both sides and the solution of the linear system of equations is provided by an explicit expression.

The iteration to compute modal function $\widehat{G}_i$, provided that $\widehat{\mathbf{u}}$ and $\widehat{G}_j$ are available for $j \neq i$, leads also to a simple equation. The sectional functional to be minimized reads

$$\begin{aligned} \mathcal{J}(\widehat{G}_i) = & \overbrace{\left[(\widehat{\mathbf{u}}^\top \mathbf{M}_u \widehat{\mathbf{u}}) \prod_{j \neq i} (\widehat{G}_j, \widehat{G}_j) \right]}^{\eta_i} (\widehat{G}_i, \widehat{G}_i) + \left\| \sum_{m=1}^n \mathbf{u}^m \prod_{i=1}^{n_p} G_i^m \right\|_{\text{Glob}}^2 \\ & - 2 \sum_{m=1}^n \underbrace{(\widehat{\mathbf{u}}^\top \mathbf{M}_u \mathbf{u}^m) \prod_{j \neq i} (\widehat{G}_j, G_j^m)}_{\xi_i^m} (\widehat{G}_i, G_i^m), \end{aligned}$$

which results in

$$\widehat{G}_i(\cdot) = \frac{1}{\eta_i} \sum_{m=1}^n \xi_i^m G_i^m(\cdot). \quad (35)$$

Note that expressions (34) and (35) provide a very simple solver for each of the alternated directions iteration. In the case of the subsequent terms of $\mathbf{U}_{\text{com}}^{\widehat{n}}$, that is for $\widehat{n} > 1$, the expressions are very similar because one has to replace only $\mathbf{U}_{\text{PGD}}^n$ by $\mathbf{U}_{\text{PGD}}^n - \mathbf{U}_{\text{com}}^{\widehat{n}-1}$. Therefore, the expressions (34) and (35) are not significantly different, just that the sums take $n + \widehat{n} - 1$ terms instead of n .

The stopping criteria in this case are identical to those presented in Section 3.4.3. Again, in the rank-one approximation, the only check that has to be performed is that the iterations have reached a stationary configuration. In fact, the same expression as in equation (31) can be straightforwardly used here taking $\widehat{\mathbf{u}}$ instead of $\mathbf{u}$ and $\widehat{G}_i$ instead of G_i .

4 Numerical examples: application to structured auxetic patterns

The generalization of PGD to structural problems is used to model a parametric architected material which is expected to exhibit auxetic properties (that is, negative Poisson ratios). The *inverted honeycomb* shaped cell is adopted here as a mechanism to generate an auxetic behavior, as introduced in [1]. The geometry of the structure to be analyzed is shown in Figure 1 and the corresponding unit-cell and its parameterization are illustrated in Figure 2. The unit-cell is formed by 8 segments (numbered with lowercase roman numbers, $i, ii, \dots, vii, viii$) modelled by beam elements and the actual shape of the cell is characterized by the following parameters:

- a : the length of the obliquely oriented beam elements ii, iv, vi and $viii$.
- b : the length of horizontal beam elements iii and vii (this parameter is set to one, $b = 1$, in the following examples).
- t : the thickness of the elements iii and vii , being $2t$ in the others.
- α : the angle of the elements ii, iv, vi and $viii$ with respect to the horizontal.

Fig. 1: 2D Architected material formed by a periodic repetition of a unit-cell, marked in red.

Fig. 2: Inverted honeycomb shaped unit-cell with its parameterization and numbering of the nodes and elements.

Thus, the $\mathbf{n}_p = 3$ free parameters are t , a and α , that is;

$$\boldsymbol{\mu} = [\mu_1 \ \mu_2 \ \mu_3]^\top = [t \ a \ \alpha]^\top.$$

The width of the cell, w in Figure 2 has to be larger than b and therefore the restriction $\cos \alpha < b/2a$ must hold.

For our parametric analyses, the parameters were chosen to range in the following intervals

$$t \in I_1 = \left[\frac{1}{200}, \frac{1}{20}\right]; a \in I_2 = [0.3, 0.7] \text{ and } \alpha \in I_3 = \left[\frac{\pi}{4}, \frac{3\pi}{4}\right].$$

That is, the solution $\mathbf{U}_{\text{PGD}}^n(\cdot)$ takes values in $\mathcal{D} = I_1 \times I_2 \times I_3$, where $\boldsymbol{\mu}$ ranges. The corresponding intervals are uniformly discretized such that the spaces V_1 , V_2 and V_3 have dimensions $\mathbf{n}_{d,1} = \mathbf{n}_{d,2} = 100$ and $\mathbf{n}_{d,3} = 500$.

The loads in all the examples are enforced via prescribed displacements (no tractions are applied on the boundary). In the analysis of the unit-cell, this includes periodicity constraints. The expressions for the parametric dependence of stiffness and mass matrices can be easily derived from standard Bernoulli beam elements. Recall that the free parameters describing the lattice structure are t , a and α . The other parameters of the beams (b , Young's modulus, depth of the rectangular cross-section) are taken to be equal to one. The non-homogeneous Dirichlet boundary conditions (nonzero prescribed displacements) are enforced in the first PGD mode, thus homogeneous essential conditions are enforced in the subsequent modes.

The tolerance η_{tol} in (30) to control the convergence of the alternated directions iteration is set to 10^{-6} , see schematic implementation in Algorithms 1 and 2 of Appendix A. To control the number of terms in the PGD expansion, the criterion selected is based on the reduction of the amplitudes β^m , $m =$

Fig. 3: Prescribed load cases for the material structure at macro-scale.

$1, 2, \dots, n$. Namely, the process is stopped at term number n if

$$\frac{\beta^n}{\max_{m=1, \dots, n}(\beta^m)} < 10^{-3}.$$

These considerations are valid for the three examples presented in the following: 1) the full parametric solution of the unit-cell for the three loading cases allowing to build up an homogenized bulk mode (Section 4.1), 2) the solution of the global parametric problem (Section 4.2) and 3) the global solution of a non fully periodic structure (Section 4.3).

4.1 Unit-cell problem using homogenization

The goal here is to exploit the generalized solution of the parametric unit-cell to extract the effective mechanical properties at the macro-scale. As it is standard in the homogenization practice, see [4, 20] and references therein, the unit-cell has to be subjected to different periodic loading conditions. In this case three loading cases are considered as described in Figure 3: the three solutions are denoted $\mathbf{U}^1 \equiv \mathbf{U}^{XX}$, $\mathbf{U}^2 \equiv \mathbf{U}^{YY}$ and $\mathbf{U}^3 \equiv \mathbf{U}^{XY}$. Analyzing the unit-cell model (Figure 2) allows computing the material effective mechanical properties through homogenization theory, which describes how these loading conditions, applied to a material structure at macro-scale, should be applied equivalently to the unit-cell by using periodicity boundary conditions at micro-scale. For further details on homogenization, we suggest the readers the works of [4, 20] and references therein. Once the three solutions are available, the effective constitutive Hooke matrix for the homogenized material reads

$$\mathbf{C}^{\text{eff}} = \begin{bmatrix} C_{11}^{\text{eff}} & C_{12}^{\text{eff}} & C_{13}^{\text{eff}} \\ C_{21}^{\text{eff}} & C_{22}^{\text{eff}} & C_{23}^{\text{eff}} \\ C_{31}^{\text{eff}} & C_{32}^{\text{eff}} & C_{33}^{\text{eff}} \end{bmatrix}, \quad (36)$$

where

$$C_{IJ}^{\text{eff}}(\boldsymbol{\mu}) = \frac{1}{wh} (\mathbf{U}^I(\boldsymbol{\mu}))^\top \mathbf{K}(\boldsymbol{\mu}) \mathbf{U}^J(\boldsymbol{\mu}), \quad \text{for } I, J = 1, 2, 3. \quad (37)$$

It is worth noting that wh stands for the area occupied by the cell. Note that both left and right hand sides of equation (37) have physical units of stiffness. On the left hand side, there are components of the 2D constitutive matrix, whereas on the right hand side we have units of energy divided by area. This corresponds to units of stiffness as well, assuming plane elasticity.

In each of the three loading cases, the results displayed are the following:

- the evolution of the modal amplitudes β^m , $m = 1, 2, \dots, n$ with and without PGD compression, see Figures 4, 7 and 10
- the shape of the first modal displacements $\mathbf{u}^1$ and $\mathbf{u}^2$, see Figures 5, 8 and 11
- the normalized parametric functions, $\tilde{G}_i^m$, for all the modes with PGD compression. In order to provide an idea of the modal amplitude, each function $\tilde{G}_i^m$ is weighted by $(\beta^m)^{\frac{1}{n_p}}$, $n_p = 3$, see Figures 6, 9 and 12.

Fig. 4: Unit-cell: load case XX , evolution of modal amplitudes β^m , $m = 1, 2, \dots, n$.

Fig. 5: Unit-cell: load case XX , shape of the first modal vectors with PGD compression (in red).

Fig. 6: Unit-cell: load case XX , normalized modal functions with PGD compression.

Fig. 7: Unit-cell: load case YY , evolution of modal amplitudes β^m , $m = 1, 2, \dots, n$.

Fig. 8: Unit-cell: load case YY , shape of the first modal vectors with PGD compression (in red).

Fig. 9: Unit-cell: load case YY, normalized modal functions with PGD compression.

Fig. 10: Unit-cell: load case XY, evolution of modal amplitudes β^m , $m = 1, 2, \dots, n$.

Fig. 11: Unit-cell: load case XY , shape of the first modal vectors with PGD compression (in red).

Fig. 12: Unit-cell: load case XY , normalized modal functions with PGD compression.

Figures 4, 5 and 6 refer to the load case XX . It is observed that in order to reach the desired accuracy, the original PGD solution requires 15 modes, which are compressed into 7, using the technique described in Section 3.5.

Figures 7, 8 and 9 refer to the load case YY , where the numbers of PGD terms is 4 without compression and 3 once compressed. In this case it is clearly seen that the second modal function in Figure 9c (blue color) governs the auxetic behaviour of the structured material for $\alpha < 90^\circ$.

Figures 10, 11 and 12 illustrate the load case XY , where the required number of modes is 6 and 4, for the raw PGD and the compressed one, respectively.

As shown in Appendix B, the analytical solution $\mathbf{U}(\boldsymbol{\mu})$ of the discrete system (1) in terms of the parameters μ_i is available. This is going to be used in the following to assess the convergence of $\mathbf{U}_{\text{PGD}}^n(\boldsymbol{\mu})$, the proposed numerical solution using PGD with compression. In order to measure the error, the parametric solution is evaluated in the Cartesian domain $I_1 \times I_2 \times I_3$ using uniformly distributed points. For intervals I_1 and I_2 , 25 points are used, while 125 points are used for I_3 . Then, the relative error of generalized displacements is computed as:

$$\epsilon_r = \frac{\sum_{k=1}^{125} \sum_{j=1}^{25} \sum_{i=1}^{25} \left\| \mathbf{U}(\mu_1^i, \mu_2^j, \mu_3^k) - \mathbf{U}_{\text{PGD}}^n(\mu_1^i, \mu_2^j, \mu_3^k) \right\|}{\sum_{k=1}^{125} \sum_{j=1}^{25} \sum_{i=1}^{25} \left\| \mathbf{U}(\mu_1^i, \mu_2^j, \mu_3^k) \right\|}, \quad (38)$$

where the superscripts i , j and k range the points in the domain where the multidimensional representation is sampled.

Fig. 13: Relative error of $\mathbf{U}_{\text{PGD}}^n(\boldsymbol{\mu})$ in the unit-cell depending on the n number of PGD modes with compression.

The results of the convergence using equation (38) for the load cases XX , YY and XY are shown in Figures 13a, 13b and 13c, respectively.

The components of the effective constitutive tensor can be constructed using the generalized displacement solutions of the unit-cell problem. For the

present example, it can be shown that the effective mechanical properties result orthotropic, see Appendix B.

One of the most relevant mechanical properties in auxetic materials is the effective Poisson's ratio. Being the material sheet orthotropic, two different Poisson's ratios - ν_{12}^{eff} and ν_{21}^{eff} - are obtained using (36) and (37), as mentioned in [6], which assumes a plane stress state and reads:

$$\begin{aligned}\nu_{12}^{\text{eff}} &= \frac{C_{12}^{\text{eff}}}{C_{22}^{\text{eff}}} \text{ and} \\ \nu_{21}^{\text{eff}} &= \frac{C_{12}^{\text{eff}}}{C_{11}^{\text{eff}}}.\end{aligned}\tag{39}$$

Therefore, in order to compute (39), the displacements solutions of the unit-cell problem for load cases XX and YY are needed.

In Figures 14a and 14b we can see the orthotropic Poisson's ratios for the parametric Cartesian domain given by $I_2 \times I_3$, taking $t = \frac{1}{40}$. We fixed t as a constant since a and α parameters are highly more relevant for Poisson's ratio variations.

The represented response surfaces are computed using the PGD solutions for the unit-cell subjected to load cases XX and YY . The approximations have 7 and 3 modes respectively, as shown when using PGD compression.

Fig. 14: Orthotropic effective Poisson's ratio using PGD.

Figure 14 shows very realistic and detailed response surfaces of the Poisson's ratio obtained by modifying parameters a and α , this latter being the most relevant parameter involved in auxetic behaviour. Furthermore, the normalized modal functions confirm such statement, including also the almost non-relevant dependence on thickness t . It is well-known that Poisson's ratio limits for isotropic materials are: $-1 < \nu < 0.5$. The values obtained for ν_{12}^{eff}

and ν_{21}^{eff} , as shown in Figure 14, do not completely verify this restriction. This is because, in the 2D-orthotropic setup, the thermodynamical consistency, see reference [19], is guaranteed by the conditions:

$$\nu_{12}^{\text{eff}} \nu_{21}^{\text{eff}} > 0 \text{ and } 1 - \nu_{12}^{\text{eff}} \nu_{21}^{\text{eff}} > 0,$$

which are truly respected by the results displayed in Figure 14.

In addition, in order to assess the error in the Poisson's ratios shown in Figure 14, an error ς_r is computed in equation (40), in a similar fashion that in (38) but considering a Cartesian domain $I_2 \times I_3$, with $\mu_1 = \frac{1}{40}$. Therefore, the accuracy of the PGD orthotropic Poisson's ratio responses is compared against the analytical expressions obtained in Appendix B. It can be seen in Figure 15 a very good agreement in the orthotropic Poisson's ratios attained by PGD.

$$\varsigma_r = \frac{\sum_{j=1}^{125} \sum_{i=1}^{25} \left\| \nu_{\text{PGD}}^{\text{eff}}(\mu_1 = \frac{1}{40}, \mu_2^i, \mu_3^j) - \nu^{\text{eff}}(\mu_1 = \frac{1}{40}, \mu_2^i, \mu_3^j) \right\|}{\sum_{j=1}^{125} \sum_{i=1}^{25} \left\| \nu^{\text{eff}}(\mu_1 = \frac{1}{40}, \mu_2^i, \mu_3^j) \right\|}, \quad (40)$$

Fig. 15: Unit-cell model relative error of Poisson's ratios between PGD vs analytical.

4.2 Periodic pattern

Here, the PGD approximation of the solution will be carried out in a periodic lattice material structure by taking into account the total number of degrees of freedom (non homogenized), instead of just a single cell. To do so, we use a periodic pattern made up from a total of 5×5 unit-cells. Two load cases are going to be considered: i) a uniaxial tensile test in X direction (Figure 16) and ii) a uniaxial tensile in Y direction (Figure 17).

Fig. 16: Periodic array loaded in X direction. Fig. 17: Periodic array loaded in Y direction.

The evolution of the modal amplitudes β^m is plotted in Figures 18 and 19 for the load cases X and Y respectively. In addition, these figures compare how modal amplitudes evolve between the standard PGD *greedy* approach and the compressed PGD one. The latter scheme shows not only a significant reduction in modes but also an improved evolution in the decreasing tendency of the modal amplitudes.

Compared with the unit-cell model, the full pattern analysis gives the possibility of a better exploration of the parameters influence in a generic structured material. This is because the boundary effects are visible in this model, whereas in the homogenized one they are cancelled by the imposed periodic constraints.

Fig. 18: Periodic pattern load case X , evolution of modal amplitudes.

Fig. 19: Periodic pattern load case Y , evolution of modal amplitudes.

In the following, a measurement of the orthotropic Poisson's ratios in the full lattice material arrangement is proposed. Due to boundary effects, the Poisson's ratio is not uniform all along the sections of the structure. Therefore, it is obtained as an average of the transversal deformations in all the corresponding boundary nodes, see Figure 20 as an example for the load case X . The estimation of $\nu_{12_{\text{PGD}}}^{\text{pat}}$ is then computed as:

$$\begin{aligned}\epsilon_{11} &= \frac{u(a, \alpha)}{w_0(a, \alpha)} = 10\% , \\ \epsilon_{22} &= \frac{h_f(t, a, \alpha) - h_0(a, \alpha)}{h_0(a, \alpha)} , \\ \nu_{12 \text{ PGD}}^{\text{pat}} &= \frac{-\epsilon_{22}}{\epsilon_{11}} = -10 \epsilon_{22} .\end{aligned}$$

Fig. 20: Measurement of $\nu_{12 \text{ PGD}}^{\text{pat}}$ in the periodic pattern subjected to load X .

The calculation of $\nu_{21 \text{ PGD}}^{\text{pat}}$ is done analogously but using load case Y instead. In Figure 21 it is shown the Poisson's ratios relative error ς_r , computed as reported in equation (40), but comparing here the full structure model against the homogenized one. Two different sizes have been considered for the full lattice material, one contains a total amount of 5×5 unit-cells whereas the other holds 10×10 unit-cells.

Fig. 21: Relative error of Poisson's ratios computed in the full lattice material structure against the homogenized one.

As expected, Figure 21 suggests that the Poisson's ratios computed in the lattice material structure compare better to the homogenized ones as long as the number of unit-cells is increased in the full structure.

4.3 Irregular pattern

In this section, the same load cases as in Section 4.2 are shown, but in this example, the irregular pattern of Figure 22 will be used, for which its analytical solution is not available due to the high complexity of the parametric result.

Fig. 22: 2D Architected material with irregular pattern.

The modal amplitudes using PGD compression can be seen in Figure 23. Results show that, as expected, the modal amplitudes decrease by increasing the number of PGD modes.

Fig. 23: Modal amplitudes using PGD with compression for a non periodic structure pattern.

The present results show a similar global behaviour when compared to the periodic pattern, despite noticeable differences in the parametric modal functions. An example of this is depicted in Figure 24 for the normalized modal functions $\hat{G}_3^m(\alpha)$.

Fig. 24: Normalized modal functions $\tilde{G}_3^m(\alpha)$ obtained with PGD compression for load case Y in the (a) periodic pattern and the (b) irregular pattern. A contrasting behaviour is observed in these oscillatory functions given the differences in the lattice material patterns.

5 Concluding remarks

The present paper presents in detail the generalization of PGD to parameterized structural problems. This tool is applied to model the macroscopic behavior of architected materials, where the parameters describe the shape and structural properties of the microstructure. This allows explicitly reproducing the response surfaces for the quantities of interest (e.g., the different orthotropic Poisson's ratios if the goal is obtaining specific auxetic properties) in terms of the design parameters. In other words, the quantities of interest are explicitly represented by computational vademecums. Consequently, the inverse problem corresponding to optimal material design is solved as a trivial post-process.

The 3D printing opens the possibility of manufacturing any of these materials for arbitrary values of the parameters and therefore this technique is currently extremely pertinent. Any optimal configuration proposed by the methodology can be easily brought into reality.

The possibility of explicitly representing the dependence of the macroscopic response as a function of the parameters describing the metamaterial cells is a promising tool. So far, the capabilities of this technology are demonstrated for 2D tessellations of seed cells with linear behavior. The extension to 3D complex structures is conceptually straightforward and opens the door to very interesting applications. Modeling the nonlinear regime is also important to properly describe the full range of applications. The nonlinear generalization is less obvious but deserves devoted research efforts.

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A . Proper Generalized Decomposition Algorithms

Data: $\mathbf{K}^k, B_i^k, \mathbf{f}^\ell, S_i^\ell$ and $\mathbf{U}_{\text{PGD}}^{n-1}(\mu)$: $\mathbf{u}^m, G_i^m$, for $m = 1, 2, \dots, n-1$,
 $k = 1, 2, \dots, n_k, \ell = 1, 2, \dots, n_f$
Result: $\mathbf{u}$ and G_i , for $i = 1, 2, \dots, n_p$
Initialize: assign values to $\mathbf{u}$ and G_i ; select a tolerance η_{tol}

```

while  $\varepsilon > \zeta$  do
    Compute  $\tilde{\mathbf{u}} = \frac{\mathbf{u}}{\|\mathbf{u}\|}$ ,  $\tilde{G}_i = \frac{G_i}{\|G_i\|}$  and  $\beta = \|\mathbf{u}\| \prod_{i=1}^{n_p} \|G_i\|$ .
    for  $k = 1 \dots n_k$  do
        Compute  $c^k := \prod_{i=1}^{n_p} \int_{I_i} B_i^k (G_i)^2 d\mu_i$ .
        for  $i = 1 \dots n_p$  do
            Compute  $d_i^k(\cdot) := \left( \prod_{\substack{j=1 \\ j \neq i}}^{n_p} \int_{I_j} B_j^k (G_j)^2 d\mu_j \right) B_i^k(\cdot)$ .
        end
        for  $m = 1 \dots (n-1)$  do
            Compute  $c^{k,m} := \prod_{i=1}^{n_p} \int_{I_i} B_i^k G_i^m G_i d\mu_i$ .
            for  $i = 1 \dots n_p$  do
                Compute  $d_i^{k,m}(\cdot) := \left( \prod_{\substack{j=1 \\ j \neq i}}^{n_p} \int_{I_j} B_j^k G_j^m G_j d\mu_j \right) B_i^k(\cdot) G_i^m(\cdot)$ .
            end
        end
    end
    end
    for  $\ell = 1 \dots n_f$  do
        Compute  $\hat{c}^\ell := \prod_{i=1}^{n_p} \int_{I_i} S_i^\ell G_i d\mu_i$ .
        for  $i = 1 \dots n_p$  do
            Compute  $\hat{d}_i^\ell(\cdot) := \left( \prod_{\substack{j=1 \\ j \neq i}}^{n_p} \int_{I_j} S_j^\ell G_j d\mu_j \right) S_i^\ell(\cdot)$ .
        end
    end
    end
    Solve  $\mathbf{u}^{\text{new}}, [\sum_{k=1}^{n_k} \mathbf{K}^k c^k] \mathbf{u}^{\text{new}} = \sum_{\ell=1}^{n_f} \mathbf{f}^\ell \hat{c}^\ell - \sum_{m=1}^{n-1} [\sum_{k=1}^{n_k} \mathbf{K}^k c^{k,m}] \mathbf{u}^m$ .
    for  $i = 1 \dots n_p$  do
        Update  $G_i^{\text{new}}(\cdot) = \frac{\sum_{\ell=1}^{n_f} (\mathbf{u}^\top \mathbf{f}^\ell) \hat{d}_i^\ell(\cdot) - \sum_{m=1}^{n-1} \sum_{k=1}^{n_k} (\mathbf{u}^\top \mathbf{K}^k \mathbf{u}^m) d_i^{k,m}(\cdot)}{\sum_{k=1}^{n_k} (\mathbf{u}^\top \mathbf{K}^k \mathbf{u}) d_i^k(\cdot)}$ .
    end
    end
    Compute  $\tilde{\mathbf{u}}^{\text{new}} = \frac{\mathbf{u}^{\text{new}}}{\|\mathbf{u}^{\text{new}}\|}$ ,  $\tilde{G}_i^{\text{new}} = \frac{G_i^{\text{new}}}{\|G_i^{\text{new}}\|}$  and  $\beta^{\text{new}} = \|\mathbf{u}^{\text{new}}\| \prod_{i=1}^{n_p} \|G_i^{\text{new}}\|$ .
    Compute  $\varepsilon = (\beta^{\text{new}})^2 + \beta^2 - 2\beta^{\text{new}}\beta (\tilde{\mathbf{u}}^\top \mathbf{M}_u \tilde{\mathbf{u}}^{\text{new}}) \prod_{i=1}^{n_p} \int_{I_i} \tilde{G}_i^{\text{new}} \tilde{G}_i d\mu_i$ .
    Update values  $\mathbf{u} \leftarrow \mathbf{u}^{\text{new}}$ ;  $G_i \leftarrow G_i^{\text{new}}$ ;  $\beta \leftarrow \beta^{\text{new}}$  and  $\zeta = \eta_{\text{tol}} \beta^{\text{new}}$ 
end

```

Algorithm 1: PGD alternated directions nonlinear solver

Data: $\mathbf{U}_{\text{PGD}}^n(\boldsymbol{\mu})$: $\mathbf{u}^m, G_i^m$, for $m = 1, 2, \dots, n$
 $\mathbf{U}_{\text{com}}^{\hat{n}-1}$: $\hat{\mathbf{u}}^{\hat{m}}, \hat{G}_i^{\hat{m}}$, for $\hat{m} = 1, 2, \dots, (\hat{n} - 1)$
Result: $\hat{\mathbf{u}}, \hat{G}_i$, for $i = 1, 2, \dots, n_p$
Initialize: assign values to $\hat{\mathbf{u}}$ and $\hat{G}_i$; select a tolerance η_{tol}

```

while  $\varepsilon > \zeta$  do
    Compute  $\tilde{\mathbf{u}} = \frac{\hat{\mathbf{u}}}{\|\hat{\mathbf{u}}\|}$ ,  $\tilde{G}_i = \frac{\hat{G}_i}{\|\hat{G}_i\|}$  and  $\psi = \|\hat{\mathbf{u}}\| \prod_{i=1}^{n_p} \|\hat{G}_i\|$ .
    Compute  $\lambda = \prod_{i=1}^{n_p} (\hat{G}_i, \tilde{G}_i)$ 
    for  $i = 1 \dots n_p$  do
        Compute  $\eta_i = (\hat{\mathbf{u}}^T \mathbf{M}_u \hat{\mathbf{u}}) \prod_{j \neq i} (\hat{G}_j, \tilde{G}_j)$ 
    end
    for  $m = 1 \dots (n - 1)$  do
        Compute  $\gamma^m = \prod_{i=1}^{n_p} (\hat{G}_i, G_i^m)$ 
        for  $i = 1 \dots n_p$  do
            Compute  $\xi_i^m = (\hat{\mathbf{u}}^T \mathbf{M}_u \mathbf{u}^m) \prod_{j \neq i} (\hat{G}_j, G_j^m)$ 
        end
    end
    end
    for  $\hat{m} = 1 \dots (\hat{n} - 1)$  do
        Compute  $\phi^{\hat{m}} = \prod_{i=1}^{n_p} \int_{I_i} \hat{G}_i \hat{G}_i^{\hat{m}} d\mu_j$ .
        for  $i = 1 \dots n_p$  do
            Compute  $\kappa_i^{\hat{m}} = (\hat{\mathbf{u}}^T \mathbf{M}_u \hat{\mathbf{u}}^{\hat{m}}) \prod_{j=1, j \neq i}^{n_p} \int_{I_j} \hat{G}_j \hat{G}_j^{\hat{m}} d\mu_j$ .
        end
    end
    end
    Update  $\hat{\mathbf{u}}^{\text{new}} = \sum_{m=1}^n \hat{\mathbf{u}}^m \left( \frac{\gamma^m}{\lambda} \right) - \sum_{\hat{m}=1}^{\hat{n}-1} \hat{\mathbf{u}}^{\hat{m}} \left( \frac{\phi^{\hat{m}}}{\lambda} \right)$ .
    for  $i = 1 \dots n_p$  do
        Update  $\hat{G}_i^{\text{new}}(\cdot) = \sum_{m=1}^n \left( \frac{\xi_i^m}{\eta_i} \right) G_i^m(\cdot) - \sum_{\hat{m}=1}^{\hat{n}-1} \left( \frac{\kappa_i^{\hat{m}}}{\eta_i} \right) \hat{G}_i^{\hat{m}}(\cdot)$ .
    end
    end
    Compute  $\hat{\mathbf{u}}^{\text{new}} = \frac{\hat{\mathbf{u}}^{\text{new}}}{\|\hat{\mathbf{u}}^{\text{new}}\|}$ ,  $\tilde{G}_i^{\text{new}} = \frac{\hat{G}_i^{\text{new}}}{\|\hat{G}_i^{\text{new}}\|}$  and  $\psi^{\text{new}} = \|\hat{\mathbf{u}}^{\text{new}}\| \prod_{i=1}^{n_p} \|\hat{G}_i^{\text{new}}\|$ .
    Compute  $\varepsilon = (\psi^{\text{new}})^2 + \psi^2 - 2\psi^{\text{new}}\psi \left( (\hat{\mathbf{u}}^{\text{new}})^T \mathbf{M}_u \hat{\mathbf{u}} \right) \left( \prod_{i=1}^{n_p} \int_{I_i} \tilde{G}_i^{\text{new}} \tilde{G}_i d\mu_j \right)$ .
    Update values  $\hat{\mathbf{u}} \leftarrow \hat{\mathbf{u}}^{\text{new}}$ ;  $\hat{G}_i \leftarrow \hat{G}_i^{\text{new}}$ ;  $\psi \leftarrow \psi^{\text{new}}$  and  $\zeta = \eta_{\text{tol}} \psi^{\text{new}}$ 
end

```

Algorithm 2: Least-Squares PGD alternated directions nonlinear solver

B . Analytical solutions

Given the unit-cell global stiffness matrix built by Bernoulli beam elements, $\mathbf{K}(\boldsymbol{\mu})$, the equations

$$\begin{aligned} \mathbf{K}(\boldsymbol{\mu}) \mathbf{U}(\boldsymbol{\mu}) &= \mathbf{0}, \\ \mathbf{C} \mathbf{U}(\boldsymbol{\mu}) &= \mathbf{Q}(\boldsymbol{\mu}), \end{aligned}$$

contain equilibrium and periodicity conditions, given a specific load case in the unit-cell. If the constraints are applied using a direct method [11], then the system unknowns can be split into released and constrained such as

$$[\mathbf{C}_R \ \mathbf{C}_C] \begin{Bmatrix} \mathbf{U}_R(\boldsymbol{\mu}) \\ \mathbf{U}_C(\boldsymbol{\mu}) \end{Bmatrix} - \{\mathbf{Q}(\boldsymbol{\mu})\} = \{\mathbf{0}\},$$

in a way that the constrained unknowns can be solved in terms of the released

$$\begin{Bmatrix} \mathbf{U}_R(\boldsymbol{\mu}) \\ \mathbf{U}_C(\boldsymbol{\mu}) \end{Bmatrix} = \underbrace{\begin{bmatrix} \mathbf{I} \\ -\mathbf{C}_C^{-1}\mathbf{C}_R \end{bmatrix}}_{\boldsymbol{\varphi}} \{\mathbf{U}_R(\boldsymbol{\mu})\} + \underbrace{\begin{Bmatrix} \mathbf{0} \\ \mathbf{C}_C^{-1}\mathbf{Q}(\boldsymbol{\mu}) \end{Bmatrix}}_{\mathbf{Q}_0(\boldsymbol{\mu})}. \quad (41)$$

The system can be solved for the released unknowns only:

$$\mathbf{K}_R(\boldsymbol{\mu}) \mathbf{U}_R(\boldsymbol{\mu}) = \mathbf{F}_R(\boldsymbol{\mu}), \quad (42)$$

where

$$\begin{aligned} \mathbf{K}_R(\boldsymbol{\mu}) &= \boldsymbol{\varphi}^T \mathbf{K}(\boldsymbol{\mu}) \boldsymbol{\varphi}, \\ \mathbf{F}(\boldsymbol{\mu})_R &= -\boldsymbol{\varphi}^T \mathbf{K}(\boldsymbol{\mu}) \mathbf{Q}_0(\boldsymbol{\mu}). \end{aligned}$$

While $\forall \boldsymbol{\mu} \in \mathcal{D} = I_1 \times I_2 \times \dots \times I_{n_p}$, $\mathbf{K}(\boldsymbol{\mu}) \in \mathbb{R}^{(n_{\text{dof}} \times n_{\text{dof}})}$, $\mathbf{K}_R(\boldsymbol{\mu}) \in \mathbb{R}^{(n_R \times n_R)}$. In the unit-cell problem, $n_{\text{dof}} = 24$ is reduced to $n_R = 13$. Now the system can be solved symbolically with the parameters $\boldsymbol{\mu} = [t \ a \ b \ \alpha]^T$.

With this, the parametric solution for case XX results:

$$\mathbf{U}_R(\boldsymbol{\mu}) = \begin{Bmatrix} u_2 \\ v_2 \\ \theta_2 \\ u_3 \\ v_3 \\ \theta_3 \\ u_4 \\ v_4 \\ \theta_4 \\ u_5 \\ v_5 \\ \theta_5 \\ \theta_6 \end{Bmatrix} = \begin{Bmatrix} \frac{b(a \cos(\alpha) - b) ((a^2 - 4t^2) \sin^2(\alpha) - a^2)}{a^2(a + 2b) - 2b(a^2 - 4t^2) \sin^2(\alpha)} \\ 0 \\ 0 \\ -\frac{(a \cos(\alpha) - b) (a^2(a + b) - b(a^2 - 4t^2) \sin^2(\alpha))}{a^2(a + 2b) - 2b(a^2 - 4t^2) \sin^2(\alpha)} \\ 0 \\ 0 \\ -\frac{(a \cos(\alpha) - b) (a^2(a + 3b) - 3b(a^2 - 4t^2) \sin^2(\alpha))}{a^2(a + 2b) - 2b(a^2 - 4t^2) \sin^2(\alpha)} \\ 0 \\ 0 \\ 0 \\ -\frac{(a \cos(\alpha) - b) (a^2(2a + 3b) - 3b(a^2 - 4t^2) \sin^2(\alpha))}{a^2(a + 2b) - 2b(a^2 - 4t^2) \sin^2(\alpha)} \\ 0 \\ 0 \\ 0 \end{Bmatrix}.$$

Figure 25 shows a scaled analytical solution for case XX , evaluated at parameters $b = 1$, $t = \frac{1}{40}$, $a = 0.5$, $\alpha = 60^\circ$ as an example.

Fig. 25: Scaled generalized displacement solution case XX .

The solution for load case YY becomes:

$$\mathbf{U}_R(\boldsymbol{\mu}) = \begin{Bmatrix} u_2 \\ v_2 \\ \theta_2 \\ \theta_3 \\ u_4 \\ v_4 \\ \theta_4 \\ u_5 \\ v_5 \\ \theta_5 \\ u_6 \\ v_6 \\ \theta_6 \end{Bmatrix} = \begin{Bmatrix} -\frac{2ab(a^2 - 4t^2)\cos(\alpha)\sin^2(\alpha)}{a^3 + a^2b + 4bt^2 + b(a^2 - 4t^2)\cos(2\alpha)} \\ a\sin(\alpha) \\ 0 \\ 0 \\ -\frac{2ab(a^2 - 4t^2)\cos(\alpha)\sin^2(\alpha)}{a^3 + a^2b + 4bt^2 + b(a^2 - 4t^2)\cos(2\alpha)} \\ 2a\sin(\alpha) \\ 0 \\ 0 \\ a\sin(\alpha) \\ 0 \\ -\frac{ab(a^2 - 4t^2)\cos(\alpha)\sin^2(\alpha)}{a^3 + a^2b + 4bt^2 + b(a^2 - 4t^2)\cos(2\alpha)} \\ a\sin(\alpha) \\ 0 \end{Bmatrix}.$$

Figure 26 shows a scaled analytical solution for case YY , evaluated at parameters $b = 1$, $t = \frac{1}{40}$, $a = 0.5$, $\alpha = 60^\circ$.

Fig. 26: Scaled generalized displacement solution case YY .

Finally, for case XY :

$$\mathbf{U}_R(\mu) = \begin{pmatrix} u_2 \\ v_2 \\ \theta_2 \\ u_3 \\ v_3 \\ \theta_3 \\ u_4 \\ v_4 \\ \theta_4 \\ u_5 \\ v_5 \\ \theta_5 \\ \theta_6 \end{pmatrix} = \left\{ \begin{array}{l} 0 \\ \frac{b(a \cos(\alpha) - b) (8t^2 \cos(\alpha)a + b(-a^2 - 2ab - 4t^2 + (a^2 + 2ab - 4t^2) \cos(2\alpha)))}{2(8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha))} \\ \frac{(a \cos(\alpha) - b) (8t^2 \cos(\alpha)a + b(-a^2 - 4t^2 + (a^2 - 4t^2) \cos(2\alpha)))}{8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha)} \\ 0 \\ -\frac{(a \cos(\alpha) - b) (16t^2a^2 + a^2b^2 - 24t^2 \cos(\alpha)ab + 2ab^3 + 4t^2b^2 - b^2(a^2 + 2ab - 4t^2) \cos(2\alpha))}{2(8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha))} \\ \frac{(a \cos(\alpha) - b) (8t^2 \cos(\alpha)a + b(-a^2 - 4t^2 + (a^2 - 4t^2) \cos(2\alpha)))}{8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha)} \\ 0 \\ -\frac{(a \cos(\alpha) - b) (16t^2a^2 + 3a^2b^2 - 40t^2 \cos(\alpha)ab + 18ab^3 + 12t^2b^2 - 3b^2(a^2 + 6ab - 4t^2) \cos(2\alpha))}{2(8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha))} \\ \frac{(a \cos(\alpha) - b) (8t^2 \cos(\alpha)a + b(-a^2 - 4t^2 + (a^2 - 4t^2) \cos(2\alpha)))}{8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha)} \\ 0 \\ -\frac{(a \cos(\alpha) - b) (32t^2a^2 + 3a^2b^2 - 56t^2 \cos(\alpha)ab + 18ab^3 + 12t^2b^2 - 3b^2(a^2 + 6ab - 4t^2) \cos(2\alpha))}{2(8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha))} \\ -\frac{2(a \cos(\alpha) - b) (-4a \cos(\alpha)t^2 + b(4t^2 + (a^2 - 4t^2) \sin^2(\alpha)))}{8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha)} \\ \frac{(a \cos(\alpha) - b) (8t^2 \cos(\alpha)a + b(-a^2 - 3ab - 4t^2 + (a^2 + 3ab - 4t^2) \cos(2\alpha)))}{8t^2a^2 + a^2b^2 - 16t^2 \cos(\alpha)ab + 5ab^3 + 4t^2b^2 - b^2(a^2 + 5ab - 4t^2) \cos(2\alpha)} \end{array} \right\}.$$

Figure 27 shows a scaled analytical solution for case XY , evaluated at parameters $b = 1$, $t = \frac{1}{40}$, $a = 0.5$, $\alpha = 60^\circ$.

Fig. 27: Scaled generalized displacement solution case XY .

At this point, it seems worth mentioning the parametric analytical solutions have the size of a unit-cell reduced by its periodic constraints. For a full structure done by repetitions of unit-cells, or a pattern with irregularities, the cost of obtaining such expressions becomes computationally unfeasible.

Finally, the components of the effective elasticity matrix are calculated from (37) and are shown in equations (43) to (48). It is worth mentioning that these results are affected by a factor of E , the Young's modulus of the constituent material, which is here omitted for the sake of clarity:

$$C_{11}^{\text{eff}}(\boldsymbol{\mu}) = -\frac{t \csc(\alpha)(a \cos(\alpha) - b) ((a^2 - 4t^2) \cos(2\alpha) + a^2 + 4t^2)}{a(a^3 + b(a^2 - 4t^2) \cos(2\alpha) + a^2b + 4t^2b)} \quad (43)$$

$$C_{22}^{\text{eff}}(\boldsymbol{\mu}) = -\frac{t \sin(\alpha) (a^3 - a(a^2 - 4t^2) \cos(2\alpha) + 4at^2 + 16t^2b)}{(a \cos(\alpha) - b)(a^3 + b(a^2 - 4t^2) \cos(2\alpha) + a^2b + 4t^2b)} \quad (44)$$

$$C_{33}^{\text{eff}}(\boldsymbol{\mu}) = -\frac{16t^3 \sin(\alpha)(a \cos(\alpha) - b)}{-b^2(a^2 + 5ab - 4t^2) \cos(2\alpha) + 8a^2t^2 + a^2b^2 - 16ab t^2 \cos(\alpha) + 5ab^3 + 4t^2b^2} \quad (45)$$

$$C_{12}^{\text{eff}}(\boldsymbol{\mu}) = \frac{t(4t^2 - a^2) \sin(2\alpha)}{a^3 + b(a^2 - 4t^2) \cos(2\alpha) + a^2b + 4t^2b} \quad (46)$$

$$C_{13}^{\text{eff}}(\boldsymbol{\mu}) = 0 \quad (47)$$

$$C_{23}^{\text{eff}}(\boldsymbol{\mu}) = 0 \quad (48)$$

Using (43), (44) and (46) the orthotropic Poisson's ratios can be calculated as follows, which assumes a state of plane stress ([6]):

$$\nu_{12}^{\text{eff}}(\boldsymbol{\mu}) = \frac{C_{12}^{\text{eff}}(\boldsymbol{\mu})}{C_{22}^{\text{eff}}(\boldsymbol{\mu})} = \frac{2(a^2 - 4t^2) \cos(\alpha)(a \cos(\alpha) - b)}{a^3 - a(a^2 - 4t^2) \cos(2\alpha) + 4at^2 + 16t^2b} \quad (49)$$

$$\nu_{21}^{\text{eff}}(\boldsymbol{\mu}) = \frac{C_{12}^{\text{eff}}(\boldsymbol{\mu})}{C_{11}^{\text{eff}}(\boldsymbol{\mu})} = \frac{a(a^2 - 4t^2) \sin(\alpha) \sin(2\alpha)}{(a \cos(\alpha) - b)((a^2 - 4t^2) \cos(2\alpha) + a^2 + 4t^2)} \quad (50)$$